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113	A slenderness based method for web crippling design of aluminium tubular sections
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## 123 Abstract

124 Existing web crippling design provisions available in European, American and Australian/New Zealand standards are based on empiric equations that differ from the approach adopted for the 125 treatment of other instabilities such as local or overall buckling which employ  $\chi$ - $\lambda$  curves. 126 Assessment of those empiric web crippling provisions based on test data available in the 127 literature and reported herein for Interior-One-Flange (IOF) and Interior-Two-Flange (ITF) 128 loading conditions has demonstrated that they provide unsafe and inconsistent predictions 129 thereby highlighting the need to develop an alternative approach. Focusing on aluminium 130 tubular sections subjected to IOF and ITF loading conditions, this article reports experimental 131 132 and numerical results that were used to develop  $\chi$ - $\lambda$  curves for web crippling design. The tubular sections were made of 6060 and 6063-T6 aluminium alloys and were manufactured by 133 extrusion. A total number of twelve tests were carried out and subsequently used to develop 134 135 and calibrate a numerical model. The measured dimensions, material properties and web

crippling loads attained are reported. After successful calibration of the numerical model, 136 parametric studies covering a wide range of slenderness and support lengths were carried out. 137 In order to derive the  $\chi$ - $\lambda$  approach three numerical analysis were performed as part of the 138 parametric study: (i) a linear elastic analysis; (ii) a plastic analyses; and (iii) a geometrical and 139 material non linear analysis. A total number of 288 numerical results were used to derive the 140 new method. Compared with European, American and Australian/New Zealand standards, the 141 142 derived  $\chi$ - $\lambda$  design method provides more accurate and reliable predictions for web crippling of aluminium tubular sections. 143

### 144 **1. Introduction**

Web crippling is a phenomenon that develops in metallic structures subjected to concentrated 145 146 transverse forces. The transverse force compresses the webs of the cross-section, which might 147 consequently yield, buckle or yield and buckle simultaneously depending on the cross-section shape and web slenderness. There are four possible web crippling load conditions as defined 148 in the North American Specification NAS (2001) and the AS/NZS 1664.1 (1997) namely 149 Interior-One-Flange (IOF), Interior-Two-Flange (ITF), Exterior-One-Flange (ETF) and 150 Exterior-Two-Flange (ETF). They are all depicted in Figure 1 where s<sub>s</sub> is the length over which 151 the transverse load is applied commonly referred to as bearing length and h<sub>w</sub> is the high of the 152 web, see Figure 2. These load conditions were first defined by Winter and Pian (1946) in the 153 154 first web crippling experimental investigation reported in the literature.

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There is extensive research available on web crippling performance of cross-sections made of cold-formed steel and stainless steel, however, the amount of web crippling studies on aluminium alloys is less extensive. With aluminium becoming more popular in the construction industry due to its lightweight properties, corrosion resistance and high strength to weight ratio, their structural performance needs to be further investigated to ultimately explore further applications. Particular attention must be placed upon the fact that its Young's modulus is one third of steel's which makes aluminium structures more susceptible to deflections and instabilities including web crippling.

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The first web crippling investigation on aluminium sections found in the literature was carried 165 out by Tryland et al. (1999). I-section and SHS extrusions were tested under the IOF and EOF 166 167 load conditions to generate well documented experimental data. Repeated tests rotating extrusions about their longitudinal axis as well as bearing lengths with circular and squared 168 shapes were also taken into consideration. Later on, Zhou and Young (2008) reported a wide 169 170 experimental programme on SHS and RHS under ITF and ETF load conditions. The 171 experimental data were used to find strength-to-yield stress ratios for normal and high strength alloys and assess the impact of the bearing length on the web crippling strength. It was found 172 that high strength alloys can achieve web crippling resistances 20% higher than normal alloys 173 and that the influence of the bearing length (i.e. the length over which the transverse load is 174 applied) on the web crippling resistance is higher for ETF loading than for ITF. In the 175 companion paper, Young and Zhou (2008) used their previous data from Zhou and Young 176 (2008) to assess the NAS (2001), AS/NZS 1664.1 (1997), AA (2005) and EN 1999-1-1 (2000) 177 178 and developed unified empirical equations for ITF and ETF that showed improved web crippling predictions. Their equations follow the same empirical principle of the design 179 provisions given in NAS (2001). In Zhou et al. (2009), the same team of researchers undertook 180 181 another experimental programme on SHS and RHS extrusions subjected to End Loading (EL) and Interior Loading (IL) which are alternative loading conditions where the specimens lay 182 over a continuous rigid bed that were first proposed by Zhao and Hancock (1992). Zhou and 183 Young (2010) also investigated the influence of perforations in SHS and RHS extrusions 184

subjected to ETF and ITF loading while Chen et al. (2015) derived new empirical equations 185 for all four loading cases. Su and Young (2018) conducted tests on stocky SHS and RHS and 186 187 proposed a new design methodology based on the continuous strength method (CSM) (Gardner 2008, Afshan and Gardner 2013, Bock et al. 2015 and Su et al. 2015). Alsanat et al. (2019) 188 reported the first experimental programme on cold-formed aluminium C-sections and found 189 that existing design provisions for both aluminium given by AS/NZS 1664.1 (1997) and cold-190 191 formed steel given by AS/NZS 4600 (2010) and by EN 1993-1-3 (2006) are unsuitable. Table 1 summarises existing research on web crippling of aluminium sections. As can be noted, to 192 193 date current design guidelines and models proposed for web crippling design are based on empirical equations. 194

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The present paper reports the results of an experimental programme on aluminium tubes 196 subjected to IOF and ITF. The tubes were manufactured by extrusion with squared corners. 197 The experimental results were combined with existing test data and used to assess existing 198 design codes and standards. The assessment presented here showed, as other researchers have 199 also highlighted, that existing web crippling provision are inaccurate and unreliable. This is 200 201 mainly because existing web crippling provisions are fully empirical. The test results were then subsequently used to develop, carefully calibrate a numerical model and perform an extensive 202 parametric study. The result of the parametric study were used to derive a new slenderness 203 based method for web crippling design employing strength curves which is the same approach 204 used for the treatment of other buckling instabilities such as local, overall buckling and patch 205 loading. Past studies have proved that slenderness based approaches are possible for web 206 crippling design and provide more reliable predictions (Duarte and Silvestre 2013, Bock and 207 Real 2014, Almatrafi et al. 2021, Zhao and Hancock 1995, Nguyen et al. 2017 and Nguyen et 208 209 al. 2020).

### 210 **2.** Experimental programme

This section reports the material and web crippling tests carried out within the scope of this article. Details of the experiments follow below along with an assessment of existing design standards.

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# **2.1 Description of the specimens**

The specimens consisted of SHS/RHS extrusions made of aluminium grades 6063T6 and 6060 215 with nominal dimensions H×B×t of the overall height H, overall width B and thickness t of 216 217  $50 \times 50 \times 3$ ,  $50 \times 50 \times 2$  and  $51 \times 51 \times 1.64$  as depicted in Figure 2. A total number of twelve specimens were tested, 6 under IOF loading and 6 under ITF loading. The actual dimensions 218 of each specimen were measured with a digital Vernier calliper and are reported in Table 2. 219 220 This table also reports the flat portion of the web h<sub>w</sub> (i.e. H-2t) to thickness ratio h<sub>w</sub>/t. All tubes were delivered in lengths of 500 mm approximately and were cut at the laboratories of the 221 University of Wolverhampton to the lengths L shown in Table 2 which are the minimum 222 lengths that meet the definition of IOF and ITF loading given in the NAS (2001) and the 223 AS/NZS 1664.1 (1997) for a bearing length s<sub>s</sub> and support length of 50 mm. The material 224 225 specification of the tubes as provided by the supplier is given in Table 3 where E is the Young's modulus,  $f_0$  is the 0.2% proof strength and  $f_u$  is the ultimate stress. 226

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## 228 2.2 Material coupon testing

Material coupon testing took place at the laboratories of the University of Birmingham in accordance with the EN ISO 6892-1:2016 (2016). Details of the coupons and testing methodology are available in Bock et al. (2021), an article on biaxial bending of aluminium tubes. Six coupon test results from Bock et al. (2021) were relevant to the scope of the present research. All six coupons showed a Ramberg-Osgood non-linear stress-strain relationship with marginal strain hardening, see Figure 3, with properties summarised in Table 4, where t is the measured thickness of the coupon, E is the Young's modulus,  $f_0$  is the 0.2% proof strength, n is the Ramberg-Osgood exponent,  $f_u$  is the ultimate stress and  $\varepsilon_u$  is the strain at ultimate stress. The cross-sectional material properties were based on mean values of each pair of coupon test results.

239 **2.** 

## 2.3 Web crippling tests

A total number of twelve web crippling tests were conducted, six of which were repeated tests 240 under IOF and six were repeated under ITF loading at the laboratories of the University of 241 Wolverhampton. The loads were applied by means of 50 mm thick steel bearing plates with a 242 bearing length s<sub>s</sub> of 50 mm. All the bearing plates were manufactured to load the full width of 243 244 the tube and were allowed to rotate as shown by the test set ups presented in Fig. 4. The flanges 245 of the tubes were not fastened to the bearing plates. In the IOF loading test, the load was applied at mid-span over the bearing length and blocks made of engineered wood (Bock and Real 2014) 246 247 were inserted in the tubes at the supports, which were 50 mm long, to ensure that web crippling failure occurs under the bearing length only. In the ITF loading tests, two identical bearings 248 with a length s<sub>s</sub> of 50 mm were positioned at mid lengths of each specimen. The transverse 249 load was applied by a 100 kN Zwick Roell testing frame incorporating load cells under the 250 hydraulic jack. Displacement control was used to drive the machine at a rate of 0.5mm/min. 251 252 The load readings and machine's crosshead vertical displacement were logged at 1 s intervals.

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All specimens failed by web crippling under the bearing length some of which are shown in Figure 5 while Table 6 presents the ultimate experimental load  $R_{w,u,test}$  achieved by the specimens.

#### 258 **2.4 Assessment of existing standards**

The experimental data generated within the scope of this article have been complemented with all existing tests from the literature on SHS/RHS tested under IOF (Tryland et al. 1999, Zhou and Young 2008, Chen et al. 2015 and Su and Young 2018) and ITF (Zhou and Young 2008, Chen et al. 2015 and Su and Young 2018) loading to assess existing design standards for structural aluminium that include a web crippling provision. The design standards taken into consideration were:

(i) the unified equation from the NAS (2001) given by Eq. (1) with coefficients C,  $C_R$ ,  $C_N$  and C<sub>h</sub>. Note that NAS (2001) does not provide coefficients for SHS/RHS and therefore, coefficients for channels for IOF loading and hat sections for ITF loading were used as they provided the best agreement with test data.

$$R_{w,Rd,NAS} = Ct^2 f_0 \sin \phi \left(1 - C_R \sqrt{\frac{r}{t}}\right) \left(1 + C_N \sqrt{\frac{s_s}{t}}\right) \left(1 - C_h \sqrt{\frac{h}{t}}\right)$$
(1)

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(ii) the equations from AA (2005) given by Eqs (2) and (3) for Interior and Exterior loading, respectively, where  $C_{w1} = 140$  mm,  $C_{w2} = 33$  mm and  $C_{w2} = 10$  mm with coefficients  $C_{wa}$  and  $C_{wb}$  given by Eqs (4) and (5). Both equations are reported herein for completeness albeit only Eq (5) was used in the present assessment.

$$R_{w,Rd,AA} = C_{wa}(s_s + C_{w1})/C_{wb} \text{ for Interior Loading}$$
(2)

$$R_{w,Rd,AA} = 1.2C_{wa}(s_s + C_{w2})/C_{wb} \text{ for Exterior Loading}$$
(3)

$$C_{wa} = t^2 \sin \theta \left( 0.46f_0 + 0.02\sqrt{Ef_0} \right)$$
(4)

$$C_{wb} = C_{w3} + r(1 - \cos\phi)$$
(5)

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(iii) the design approach from the EN 1999-1-1 (2000) given by Eqs. (6)-(14). This approach
is currently applicable to plate girders and therefore the original nomenclature has been adapted

to SHS/RHS. In this method a is the distance between transverse stiffeners taken as the full
length of the specimen L if no stiffeners are provided.

$$R_{w,Rd,EN1999-1-1} = \chi_F l_y t f_0 / \gamma_{M1} \tag{6}$$

$$\chi_F = 0.5/\lambda_F \tag{7}$$

$$\lambda_F = \sqrt{l_y t f_0 / F_{cr}} \tag{8}$$

$$F_{cr} = 0.9k_F E t^3 / h_w \tag{9}$$

$$k_F = 6 + 2(h_w/a)^2 \text{ for IOF loading}$$
(10)

$$k_F = 3.5 + 2(h_w/a)^2 \text{ for ITF loading}$$
(11)

$$l_y = s_s + 2t \left( 1 + \sqrt{m_1 + m_2} \right) but \ l_y \le a \ for \ IOF \ and \ ITF$$
(12)

$$m_1 = B/t \tag{13}$$

$$m_2 = 0.02(h_w/t)^2$$
 if  $\lambda_F > 0.5$  otherwise  $m_2 = 0$  (14)

279

(iv) the procedure given in EN 1993-1-3 (2006) for cold-formed steel which has also been 280 adopted in EN 1999-1-4 (2009), the part for cold-formed structural sheeting, given by Eq. (15). 281 282 In this equation the value of  $\alpha$  depends on the clear distance e between two opposing transverse loads (i.e. the support and the applied load). For  $e \le 1.5 h_w \alpha = 0.075$  while for  $e > 1.5 h_w \alpha = 0.15$ 283 and as a general rule the former applies to ITF loading and the latter to IOF loading. EN 1999-284 1-4 (2009) refers to cases where  $e \le 1.5h_w$  as Category 1 and cases of  $e > 1.5h_w$  as Category 2. 285 For Category 1  $l_a=s_s$  but  $\leq 40$  mm while for Category 2  $l_a=s_s$  for symmetric cases where the 286 287 transverse shear force on either side of the transverse load is the same.

$$R_{w,Rd,EN1999-1-4} = \alpha t^2 \sqrt{f_o E} \left(1 - 0.1\sqrt{r/t}\right) \left(0.5 + \sqrt{0.02 \, l_a/t}\right) (2.4 + (\phi/90)^2) / \gamma_{M1}$$
(15)

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In Eqs (1), (5) and (15) r is the internal bend radius of the corners of the cross-section taken as 0 for extruded sections as the corners are squared. For the assessment, all partial safety factors  $\gamma_{M1}$  were set to one and the predicted values by Eqs (1-3), (6) and (15), which provide the

resistance per web, were multiplied by the number of webs of the cross-section (i.e. 2). The 292 results are presented in Figures 6 and 7 for IOF and ITF loading respectively, where the 293 experimental  $(R_{w,exp})$  to predicted ratio  $(R_{w,pred})$  is plotted against the slenderness ratio  $h_w/t$ . The 294 assessment shows data falling both on the safe side (i.e R<sub>w.exp</sub>/ R<sub>w.pred</sub>>1) and the unsafe side 295 with high scatter highlighting the need for an alternative approach to web crippling design. 296 Statistical results are presented in Table 5, with regards to mean and coefficient of variation 297 298 (COV), which show that EN 1999-1-4 (2009) provides little agreement with experimental data for both IOF and ITF loading. The most accurate provision is that given by AA (2005) for IOF 299 300 loading and NAS (2001) for ITF loading. It is also noted for ITF loading, EN 1999-1-1 (2000) shows less scatter but higher mean (i.e. more conservative). These results strengthen the need 301 to develop an alternative approach for web crippling design. 302

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# 3. Finite Element modelling

The finite element (FE) software package Simulia Abaqus 2020 was used to simulate the structural behaviour of aluminium alloy tubes subjected to web crippling under the IOF and ITF loading conditions. Further details about the modelling approach to simulate web crippling are provided in this section

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### **310 3.1 Geometry discretization**

The geometry of the tubes was modelled by using 3D deformable shell elements and discretised using the doubly curved four-noded shell elements with reduced integration S4R. This element has six degrees of freedom at each node and has been successfully used in modelling of web crippling behaviour of aluminium tubes (Zhou and Young 2010, Chen et al 2015, Su and Young 2018 and Castaldo et al. 2016). The cross-section size was based on centreline dimensions  $h \times b$ , see Figure 2. Due to symmetry about the vertical axis for both IOF and ITF loading, half of the tube was modelled. A mesh convergence study was also performed which revealed that an element size of  $2 \text{ mm} \times 2 \text{ mm}$  provides accurate modelling results within reasonable computational time.

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## 321 **3.2 Material modelling**

The FE models incorporated both elastic and plastic parts of the stress-strain curve. The elastic part was modelled by the measured Young's modulus E and using a Poisson's ratio coefficient of 0.33. For the plastic part, the measured stress  $\sigma_{nom}$  and strain  $\varepsilon_{nom}$  were converted into true stress ( $\sigma_{true}$ ) and true plastic strain ( $\varepsilon_{pl,true}$ ) by using Eq (16) and Eq. (17), respectively, and subsequently incorporated into the models. Isotropic behaviour was specified.

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$$\sigma_{true} = \sigma_{nom} (1 + \varepsilon_{nom}) \tag{16}$$

$$\varepsilon_{pl,true} = ln(1 + \varepsilon_{nom}) - \frac{o_{true}}{E}$$
(17)

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## **329 3.3 Boundary conditions and interactions**

The boundary conditions implemented in the numerical model were the same as those used in the tests. All supporting plates and bearing lengths were modelled by using 3D analytical rigid shells. Reference points (RPs) located in the centre were assigned to all rigid shells to which boundary conditions were applied as shown in Figures 8 and 9 for IOF and ITF, respectively. In these figures U1, U2 and U3 are displacements corresponding to the x, y and z directions respectively while UR1, UR2 and UR3 are rotations about the x-x, the y-y and z-z axes. For IOF loading models the supports were allowed to rotate about the cross-sectional horizontal

axis only and the rest degrees of freedom were restrained. At one of the ends, the longitudinal

displacement was also allowed. All degrees of freedom were restrained at the reference pointof the bearing length except vertical displacement.

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For ITF loading models, all degrees of freedom of the bottom bearing plate were restrainedwhile only vertical displacement was allowed at the top bearing plate.

In both IOF and ITF loading, surface-to-surface contact interaction was applied to the regions of the tube in contact with the bearing lengths and supports as shown in Figures 8 and 9. The 3D rigid plates were defined as "Master surface" while the regions of the tube in contact were defined as "Slave surface". Contact was modelled by using hard normal behaviour and penalty isotropic tangential behaviour with a friction coefficient of 0.6. The element thickness of the tube was excluded and adjustment of the slave surface was allowed only to remove overclosure.

To apply the load in the FE models, a vertical displacement was defined at the RPs. To solve the models, the general static method in two steps allowing for geometric nonlinearities was used. The vertical displacements introduced in the first and second step were 0.1 mm and 20, respectively. It was observed that introducing the displacement in two steps removed numerical convergence issues. Like in other numerical web crippling investigations, initial imperfections or residual stresses were not modelled as they do not have any effect on the web crippling response (Ren et al. 2006, Zhou and Young 2007 and Bock et al. 2013).

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# 3.4 Validation of the numerical model

The reliability of the FE model was assessed by comparing the experimental and numerical ultimate loads and failure modes. The failure modes achieved by the models are shown in Figure 10 where it is observed that they replicate the same failure modes achieved during testing shown in Figure 5. Table 6 presents the ultimate experimental load  $R_{w,u,test}$ , the ultimate numerical to test load ratio  $R_{w,u,FE}/R_{w,u,test}$  as well as the mean value for that ratio and 362 coefficient of variation. The comparison shows a mean ratio very close to one and relatively
363 small COV for both IOF and ITF loading models thereby enabling to conclude that the
364 developed numerical model is accurate and reliable.

**365 3.5 Parametric studies** 

Following validation of the developed numerical model, parametric studies were carried out in 366 order to extend structural performance data on aluminium SHS/RHS subjected to IOF and ITF 367 web crippling loading conditions. Two geometries namely SHS and RHS, four thicknesses, 368 369 three bearing lengths and two alloys were considered. The centreline dimensions  $h \times b$  for SHS and RHS were  $50 \times 50$  and  $100 \times 50$ , respectively, while the thicknesses were 5 mm, 3 mm, 2 370 mm and 1 mm. The ratio of h/t ranged therefore from 9 to 49. The bearing lengths were 25 371 372 mm, 50 mm and 75 mm. The length of SHS was 340 mm and 240 mm for IOF and ITF loading 373 respectively, while RHS were 490 and 390 mm long for IOF and ITL loading, respectively. The aluminium alloys considered had the following material properties: (i) E = 66 GPa,  $f_0 =$ 374 300 MPa, n=35,  $f_u = 339$  MPa and  $\varepsilon_u = 0.084$  and (ii) E = 67.55 GPa,  $f_0 = 198$  MPa, n=28.5,  $f_u$ 375 = 229 MPa and  $\varepsilon_u$ =0.078. Note that these material properties were also used in Bock et al. 376 (2021) and are based on average values of tensile coupon testing. The following section 377 describes how these models were used to develop a new slenderness based approach for web 378 379 crippling design of aluminium SHS/RHS.

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## 4. Development of the slenderness based $\chi(\lambda)$ method

382 **4.1 Basis of the method** 

Slenderness based or  $\chi(\lambda)$  method is a design approach that finds the ultimate load by means of a strength curve  $\chi$  that is a function of the slenderness  $\lambda$  as given by Eqs. (18) – (20) where A and B are dimensionless coefficients requiring calibration. On this basis, the ultimate web

crippling load R<sub>w,u</sub> can therefore be found involving the buckling R<sub>w,cr</sub> and plastic R<sub>w,pl</sub> load. 386 To determine R<sub>w,u</sub>, R<sub>w,cr</sub> and R<sub>w,pl</sub> three types of numerical analyses were carried out on the 387 geometries described in parametric studies section: (i) a material and geometrical non-linear 388 analysis to obtain R<sub>w,u</sub> (ii) a linear elastic buckling analysis to obtain R<sub>w,cr</sub> and (iii) a first order 389 plastic analysis (i.e. non linear material without considering geometrical nonlinearities) to 390 obtain R<sub>w,pl</sub>. Note that the elastic R<sub>w,cr</sub> and plastic R<sub>w,pl</sub> loads can only be determined 391 392 numerically. A total number of 144 numerical results were generated and used to derive predictive models for  $R_{w,cr}$ ,  $R_{w,pl}$  and  $\chi(\lambda)$  functions for both IOF and ITF loading. 393

$$R_{w,u} = \chi R_{w,pl} \tag{18}$$

$$\chi = \frac{A}{\bar{\lambda}^B} \le 1 \tag{19}$$

$$\lambda = \sqrt{\frac{R_{w,pl}}{R_{w,cr}}} \tag{20}$$

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# **4.2 Linear elastic buckling analysis**

Linear elastic analyses were performed to obtain buckling shapes and associated eigenvalues. 396 The load taken as the critical buckling load R<sub>w,cr</sub> was the lowest elastic buckling load pertinent 397 to web buckling. Because contact interaction is incompatible with buckling analysis, for IOF 398 models the interaction between the specimen and supporting plates was simulated by using tie 399 400 type constraint and the bearing plate was simulated with a pressure load. For ITF models, the interaction between the bearing plates and specimen was simulated by using tie type constraint 401 and the load as a concentrated force applied at the reference point of the top bearing plate. To 402 obtain R<sub>w,cr</sub> the lowest eigenvalue pertinent to web buckling was multiplied by the value of the 403 applied load. Note that for IOF loading, the total magnitude of the pressure load was used. 404 405 Some examples of web buckling modes are shown in Figures 11 and 12 for bearing lengths of 25 and 75 mm, respectively, which indicate that the web buckling shape does not significantlychange with varying bearing length.

408

The numerical results from linear elastic buckling analyses R<sub>w,cr,FE</sub> were used to derive an 409 analytical model that predicts the elastic buckling load R<sub>w,cr</sub>. The model namely R<sub>w,cr,pred</sub> is 410 given in Eq. (21) which is stems from classical elastic theory of instability of a plate loaded 411 with a concentrated in-plane force at the edge where the dimensionless buckling 412 coefficient k<sub>F</sub> may be derived for a given plate geometry with certain boundary conditions. 413 Same approach was also used in Bock and Real (2014) and Almatrafi et al. (2021). Coefficients 414 k<sub>F</sub> were therefore derived herein by using multilinear regression analysis considering relevant 415 geometrical variables and are given by Eqs. (22) and (23) for IOF and ITF, respectively. A 416 graphical comparison between predicted and numerical values is presented in Figure 13 with 417 statistics given in Table 7. The comparison shows excellent agreement between numerical and 418 predicted data with mean values close to unity and reasonable COVs. Note that the predictive 419 models given in Eqs. (21) and (22) provide the elastic buckling load of the cross-section and 420 not of a single web. 421

$$R_{w,cr,pred} = k_F \frac{\pi^2 E t^3}{12(1-\nu^2)h}$$
(21)

For IOF loading (22)  

$$k_F = 0.003 h/b + 0.006 s_s/h + 4.3 \times 10^{-6} s_s/t - 0.008 t/h - 0.114 (s_s/L)^2$$

For ITF loading (23)  

$$k_F = 0.134 \, s_s / L + 1.24 \times 10^{-5} \, s_s / t - 0.7 (s_s / L)^2 + 0.004 (s_s / h)^2 + (s_s / L)^3$$

#### 423 **4.3 First order plastic analysis**

The modelling assumptions used to obtain the web crippling ultimate load R<sub>w.u</sub>, apart from 424 geometrical nonlinearities, were adopted in first-order plastic analysis to obtain the plastic load 425  $R_{w,pl}$ . The load – displacement response obtained from first order plastic analysis is shown in 426 Figure 14 for some specimens whilst highlighting the influence of the bearing length and aspect 427 428 ratio. The plotted displacement corresponds to the vertical translation of the centre of the top flange at midspan (i.e. where the RP was defined). From Figure 10 (b) it is observed that the 429 plastic load achieved by SHS is the same as its RHS counterpart. In addition to this, most of 430 curves did not reach a clear, well-defined maximum and kept increasing with reducing slope. 431 Same trend was observed in past studies on other cross-sections and materials (Duarte and 432 Silvestre 2013, Bock and Real 2014 and Almatrafi et al. 2021). To obtain the plastic load R<sub>w,pl</sub>, 433 the graphical method proposed by Dos Santos et al. (2018) was used which assumes that the 434 plastic load is attained when the slope of the load-deflection curve is 1% of the initial stiffness. 435 Other graphical methods to determine plastic loads are also available such as the Southwell 436 plot (Southwell 1932), the convergence indicator plot (Horne and Merchant 1965) and the 437 tangent stiffness plot (Doerich and Rotter 2011), which were found to provide similar 438 predictions to Dos Santos et al. (2018). Some plastic mechanism developed are shown in Figure 439 15. 440

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The numerical results from plastic analyses  $R_{w,pl,FE}$  were used to derive an analytical model that predicts the plastic load  $R_{w,pl}$ . The model namely  $R_{w,pl,pred}$  is given in Eq.(24) which is similar to the model employed by EN 1999-1-1 (2000) to predict plastic loads in the provision dealing with concentrated transverse forces which is based on plastic line theory. It is worth pointing out that this model does not contain the internal radius r and that other models for

plastic loads are available for cross-sections with rounded corners and materials such as cold 447 formed channels (Duarte and Silvestre 2013), stainless steel hat sections (Bock and Real 2014), 448 cold-formed sigma sections (Almatrafi et al. 2021), cold-formed SHS and RHS (Zhao and 449 Hancock 1995) and cold-formed lipped channels (Nguyen et al. 2020). In Eq. (24), ly is the 450 yield-prone effectively loaded length that can be calibrated from geometrical and mechanical 451 properties. Equations for l<sub>y</sub> were derived by using multilinear regression analysis considering 452 453 relevant geometrical variables for IOF and ITF loading and are given by Eqs. (25) and (26), respectively. A graphical comparison between predicted and numerical values is presented in 454 455 Figure 16 with statistics given in Table 7. As can be seen, the comparison shows excellent agreement between numerical and predicted data with mean values close to unity and 456 reasonable COVs. Note the predictive models given in Eqs. (25) and (26) provide the plastic 457 load of the cross-section and not of a single web. 458

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$$R_{w,pl,pred} = l_y t f_0 \tag{24}$$

For IOF loading

$$l_v = 2.22s_s + 0.92h + 25.82b/h + 1.16t + 0.52b/t - 613.80s_s/L - 0.65h/t \le L$$

(25)

(26)

For ITF loading

$$l_y = 2.73s_s + 0.54h + 46.59 \, b/h - 0.044 \, h/t - 1.11 \, b/t + 1.73 \, s_s/t \le L$$

461

## 462 **4.4 Proposed slenderness based** $\chi(\lambda)$ equations for IOF and ITF

463 Having obtained the ultimate web crippling load  $R_{w,u}$ , the buckling load  $R_{w,cr}$  and plastic load

464 R<sub>w,pl</sub> numerically, the resulting  $\chi$ - $\lambda$  plot shown in Figure 17 for (a) IOF and (b) ITF loading was

obtained and first slenderness  $\chi(\lambda)$  based equations were derived by using regression analysis. 465 Figure 17 also shows available curves for cold formed (CF) channels (Duarte and Silvestre 466 2013) and cold-formed sigma sections (Almatrafi et al. 2021) as well as stainless steel hat 467 sections (Bock and Real 2014). Note that other  $\chi(\lambda)$  curves with similar coefficients are also 468 available in Nguyen et al. (2020) for cold-formed lipped channels but have not been added in 469 Figure 17 for simplicity. The results show for IOF loading that the data follows a trend whereby 470 471  $\chi$  slowly decreases with increasing  $\lambda$  following an exponential law and with low scatter. For ITF loading a family of  $\chi$ - $\lambda$  curves is observed for various s<sub>s</sub>/h ratios. For both loading cases, 472 473 it is observed that the proposed  $\chi(\lambda)$  in Duarte and Silvestre (2013), Bock and Real (2014) and Almatrafi et al. (2021) are not suitable for extruded aluminium SHS/RHS. The results also 474 show that SHS/RHS aluminium sections achieve a higher  $\chi$ - $\lambda$  relationship for IOF loading than 475 existing data on other sections (i.e. channels, sigma sections and hat sections) and materials 476 (i.e. stainless steel and cold-formed steel). 477

478

The  $\chi(\lambda)$  curves shown in Figure 17 are the best fit trend lines that could be used to predict 479 ultimate web crippling loads R<sub>w,u</sub> of any aluminium SHS/RHS when elastic buckling R<sub>w,cr</sub> and 480 plastic R<sub>w,pl</sub> loads are determined numerically. As predictive models for these loads have been 481 developed in sections 4.2 and 4.3, the final proposed  $\chi(\lambda)$  curves incorporating R<sub>w,cr,pred</sub> and 482 483 R<sub>w,pl,pred</sub> from Eqs (21-26) are shown in Figure 18 and are given by Eqs. (27) and (28). Note that for ITF loading,  $\gamma(\lambda)$  is expressed as a continuous function of s<sub>s</sub>/h where coefficients A and 484 B from the general form as given in Eq. (1), have been substituted by the best fit lines shown 485 486 in Figure 19. Figure 18 shows that after the incorporation of the derived predictive models, the  $\chi$ - $\lambda$  values do not change significantly and best fit trend lines are very similar to those based on 487 488 numerical values.

For IOF loading

$$\chi = \frac{1.03}{e^{0.47\lambda}} \le 1$$

 $(\mathbf{77})$ 

(28)

For ITF loading

$$\chi = \frac{0.338 + 0.079 \, s_s/h}{\lambda^{0.863 - 0.056 s_s/h}} \le 1$$

490

491

## 492 5. Validation of the proposed $\chi(\lambda)$ method against test and FE data

Taking into consideration experimental data reported in (Zhou and Young 2008, Chen et al. 493 2015 and Su and Young 2018) as well as tests and generated FE models presented herein, the 494 proposed  $\chi(\lambda)$  curves for IOF and ITF loading given in Eqs (27) and (28) used in conjunction 495 496 with the derived models for the elastic critical load R<sub>w,cr,pred</sub> and plastic load R<sub>w,pl,pred</sub> as given by Eqs. (21)-(26) to determine the cross-sectional web crippling resistance  $R_{w,u}$  are validated. 497 To this end, the numerical R<sub>w,u,FE</sub> and experimental R<sub>w,u,test</sub> resistances of the cross-section were 498 499 divided by the predicted values by the proposed method R<sub>w,u,pred</sub>. Subsequently, mean and COV 500 values of the ratios were determined for various data sets which are presented in Table 8. 501 Compared to the statistics from the assessment of existing design standards presented in Table 502 5, it is observed that the mean and COV values of the predictions by the proposed method significantly improve. A graphical comparison between numerical/FE and predicted values is 503 presented in Figure 20 showing that both experimental and FE data sets fall closer to the 45 504 505 degree line.

506

507

A series of tests have been performed on extruded aluminium SHS/RHS under IOF and ITF 509 loading conditions. The test results were complemented with existing test data from literature 510 (Tryland et al. 1999, Zhou and Young 2008, Chen et al. 2015 and Su and Young 2018) to show 511 that the current standards for web crippling design for structural aluminium are very inaccurate 512 513 and unreliable. The test results were also used to develop and calibrate a numerical model upon which to base the development of a slenderness based approach employing strength curves  $\chi(\lambda)$ 514 for web crippling design of extruded SHS/RHS underIOF and ITF loading. To achieve this, 515 three types of analyses were undertaken: (i) a liner elastic analysis to determine critical loads 516 R<sub>w.cr</sub>, (ii) a first order plastic analysis considering perfect geometry and material nonlinearities 517 518 to find plastic loads R<sub>w,pl</sub> and (iii) a full nonlinear analysis to obtain web crippling ultimate loads R<sub>w.u.</sub>. It was found that sections with different aspect ratios h/b reach same plastic loads 519 R<sub>w,pl</sub>. The numerical values for R<sub>w,cr</sub> and R<sub>w,pl</sub> were used to derive the predictive models for 520 521 such loads as presented in Eqs. (21)-(26) and find  $\chi(\lambda)$  curves.  $\chi$  and  $\lambda$  were found as shown by Eqs. (2) and (3). For IOF loading it was observed that a single  $\chi(\lambda)$  could capture the data 522 accurately, while for ITF loading a family of  $\chi(\lambda)$  curves that depend on the s<sub>s</sub>/h ratio would be 523 needed.  $\chi$ - $\lambda$  values by using the developed predictive models from Eqs. (21)-(26) were finally 524 used to accurately calibrate the  $\chi(\lambda)$  curves given by Eqs. (27) and (28). The approach 525 526 developed within the scope of this article has led to significantly improved and reliable predictions of the web crippling resistance of extruded aluminium SHS/RHS under IOF and 527 ITF loading. 528

529 The research presented in this article has yet again proved that slenderness based approaches 530 have potential to be adopted for web crippling design. Further research on additional cross-531 section shapes and materials is underway to extend this design approach.

## 533 Data Availability Statement

Some or all data, models, or code that support the findings of this study are available from thecorresponding author upon reasonable request.

536

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	a .:	4 11	<b>T</b> 1	
Reference	Section	Alloy	Load case	Relevant Contribution
	SHS		26 IOF	Circular and rectangular bearings.
Tryland et al. (1999)	I-sec	AA6082-T6	26 EOF	Specimens were rotated about the longitudinal axis and tested three times. Investigation
				reports average values results.
				Testing and successful development of FE. Web slenderness from 6.3 to 74.5 was
Zhan and Vanna	DUC	(0 <i>C</i> 2 T5	76 ETF	investigated. It was observed that the web crippling strength to yield stress ratio for 6063-
Zhou and Young	RHS	0003-15	74 ITF	T5 is 20% larger than for 6061-T6 and that the web crippling strength for ETF loading
(2008)	SHS	6061-16		condition increases faster than those
				for ITF loading condition as the bearing length increases
				Development of two different unified equations based on experimental data. Assessment of
Young and Zhou	RHS	Aluminium		NAS (2001), AS/NZS 1664.1 (1997), AA (2005) and EN 1999-1-1 (2000). It was observed
ReferenceSectionTryland et al. (1999)I-secZhou and Young (2008)RHS SHSYoung and Zhou (2008)RHS SHSZhou et al. (2009)SHSZhou and Young (2010)RHS SHSZhou et al. (2009)SHSChen et al. (2015)SHSSu and Young (2018)RHS SHSAlsanat et al. (2019)C-sections	SHS	6063-T5	-	that the web crippling strength is correlated to the N/h ratio where N is the bearing length
	6061-T6		and h is the denth of the flat nortion of the web	
				Assessment of the $\Delta \Delta$ (2005) and EN 1999-1-1 (2000) which were found either unsafe or
Thou at al. $(2009)$	545	6061 T6	30 EL	overly conservative. Critical values of web slenderness were also proposed beyond which
Tryland et al. (1999)         Zhou and Young (2008)         Young and Zhou (2008)         Zhou et al. (2009)         Zhou and Young (2010)         Chen et al. (2015)         Su and Young (2018)	5115	0001-10	34 IL	web buckling governs. EL and IL loading first proposed in Theo and Hancock (1992)
	DUC			Web originaling strength reduction factor equations were proposed for the
Zhou and Young	6063-T5		ETF	ETE and ITE loading and litigate. The effect of wind stress on the such healthing
(2010)	SHS	6061-T6	ITF	ETF and TTF loading conditions. The effect of yield stress on the web buckling
Young and Zhou (2008)RHS SHSAlt 60 60Zhou et al. (2009)SHS60Zhou and Young (2010)RHS SHS60Chen et al. (2015)SHSNotSu and Young (2018)RHS SHS60				strength was also discussed in this paper.
Chen et al. (2015)	SHS	Not reported	12 IOF; 12 EOF	Recalibration of existing web crippling design provisions for all load cases.
	5115	rior reponted	12 ITF; 12 ETF	Recultivition of chisting web emploing design provisions for an four cuses.
Su and Young (2018)	RHS	6063-T5	6 IOF; 6 EOF	Adaptation of the CSM for web crippling design of stocky tubular extrusions
Su and Toung (2018)	SHS	6061-T6	11 ITF; 11 ETF	Adaptation of the CSW for web emploing design of stocky tubular extrusions.
				First experiments on cold-formed aluminium C-sections. Assessment of design standards
				for cold-formed steel and aluminium including AS/NZS 1664.1 (1997), AS/NZS 4600
Alsanat et al. (2019)	C-sections	5052H36	20 ETF: 20 ITF	(2010) and EN 1993-1-3 (2006) as well as Sundararajah et al. (2017) modifications for
× ,			,	cold-formed steel channels. It was found that the design strengths predicted by the
				aforementioned specifications are unconservative and unsafe.

# **Table 1.** Relevant research on aluminium alloy sections subjected to web crippling

Cross-section	Test	Mill grade	H (mm)	B (mm)	t (mm)	h <sub>w</sub> /t	L (mm)
50×50×3-1	IOF	6060	50.05	50.05	2.90	15.26	310
50×50×3-2	IOF	6060	50.06	50.06	2.91	15.20	310
50×50×2-1	IOF	6060	49.95	49.92	1.99	23.09	310
50×50×2-2	IOF	6060	49.95	49.92	1.99	23.09	310
51×51×1.64-1	IOF	6063T6	51.07	51.09	1.68	28.41	310
51×51×1.64-2	IOF	6063T6	51.06	51.08	1.68	28.40	310
50×50×3-1	ITF	6060	50.07	50.04	2.88	15.39	260
50×50×3-2	ITF	6060	50.05	50.05	2.88	15.38	260
50×50×2-1	ITF	6060	49.90	49.90	1.95	23.59	260
50×50×2-2	ITF	6060	49.95	49.95	1.95	23.62	260
51×51×1.64-1	ITF	6063T6	51.08	51.12	1.66	28.80	260
51×51×1.64-2	ITF	6063T6	51.09	51.15	1.66	28.81	260

Table 2. Measured dimensions of the tested cross-sections

Cross-section	Mill grade	E (GPa)	f <sub>0</sub> (MPa)	fu (MPa)
50×50×3	6060	70	170	260
50×50×2	6060	69.5	150	195
51×51×1.64	6063T6	69.5	170	215

**Table 4.** Relevant tensile coupon test results extracted from Bock et al. (2021)

Coupon	t (mm)	E (GPa)	fo (MPa)	n	fu (MPa)	Eu (%)
51×51×1.64-1	1.64	66.00	219	40	239	6.76
51×51×1.64-2	1.64	66.00	207	40	225	6.83
50×50×2-1	1.97	67.20	213	35	235	7.34
50×50×2-2	1.97	67.20	210	35	232	6.82
50×50×3-1	3.00	67.80	208	30	240	7.40
50×50×3-2	3.00	67.30	188	27	218	8.36

Load case	statistics	EN 1999-1-4	EN 1999-1-1	AA	NAS
		(2009)	(2000)	(2005)	(2001)
IOF	Mean	2.067	0.828	1.096	0.878
	COV	0.566	0.326	0.440	0.449
ITF	Mean	5.664	1.313	1.431	1.106
	COV	0.572	0.286	0.426	0.416

	]	IOF		ITF
<b>Cross-section</b>	Rw,u,test (kN)	$R_{w,u,FE}/R_{w,u,test}$	R <sub>w,u,test</sub> (kN)	$R_{w,u,FE}/R_{w,u,test}$
50×50×3-1	50	0.95	73	0.99
50×50×3-2	49	0.98	73	0.99
50×50×2-1	31	0.95	41	1.06
50×50×2-2	31	0.96	42	1.04
51×51×1.64-1	27	0.97	34	0.90
51×51×1.64-2	26	1.00	33	0.91
Mean		0.97		0.98
COV		0.020		0.066

# Table 7. Comparison between numerical values and derived predictive models

	IC	)F	ľ	ſF
Statistics	Rw,cr,FE/ Rw,cr,pred	Rw,pl,FE/ Rw,pl,pred	Rw,cr,FE/ Rw,cr,pred	Rw,pl,FE/ Rw,pl,pred
Mean	0.99	0.99	0.96	0.99
COV	0.041	0.069	0.062	0.041

	Data set	Sample size	Mean	COV	
IOF	FE	51	0.94	0.133	
	Tests	24	1.05	0.191	
	FE+Tests	75	0.99	0.168	
ITF	FE	51	1.01	0.110	
	Tests	103	0.98	0.268	
	FE+Tests	154	0.99	0.224	

**Table 8.** Statistics of the validation against tests and FE results



Figure 1. The four web crippling load conditions



Figure 2. Cross-sectional notation



Figure 3. Material stress-strain relationship of tested coupons extracted from [90]



Figure 4. IOF loading test set up (a) ITF loading test set up (b)



(a) (b) Figure 5. Web crippling failure modes for (a) IOF loading and (b) ITF loading tests



Figure 6. Assessment of existing design standards for IOF loading



Figure 7. Assessment of existing design standards for ITF loading



Figure 8. Interactions and boundary conditions of IOF loading models



Figure 9. Interactions and boundary conditions of ITF loading models



(a) (b) **Figure 10.** Failure modes achieved by models for (a) IOF loading and (b) ITF loading



RHS under IOF and (d) RHS under ITF with a bearing length of 75 mm







Figure 14. Load-displacement response from first order plastic analysis for (a) IOF loading and (b) ITF loading





Figure 15. Plastic mechanisms developed



**Figure 16.** Comparison between plastic loads numerically obtained R<sub>w,pl,FE</sub> and predicted by the derived model R<sub>w,pl,pred</sub> for (a) IOF loading and (b) ITF loading



**Figure 17.** Numerical results plotted in a  $\chi(\lambda)$  space for (a) IOF and (b) ITF loading



Figure 18.  $\chi(\lambda)$  curves incorporating predictive models  $R_{w,cr,pred}$  and  $R_{w,pl,pred}$  for (a) IOF and (b) ITF loading



Figure 19. Coefficients A and B from  $\chi(\lambda)$  model as a function of  $s_s/h$  for ITF loading



**Figure 20.** Comparison between FE/experimental ultimate web crippling resistances and predicted ones by the proposed method for (a) IOF and (b) ITF loading